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Airborne-Infection ISOLATION Design of a Specialized



Design of a Specialized Airborne-Infection-Isolation Suite

Lessons learned from the design of new National Institutes of Health facility

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A biocontainment patient-care unit (BPCU) is a facility digned and operated to maximize patient care with appropriate infection-control practices and procedures. A BPCU is secure, is physically separated from other patient-care areas, and has special air-handling systems.¹

The National Institutes of Health's new Special Clinical Studies Unit (SCSU) in Bethesda, Md., is a rughly specialized BPCU, housing patients with extremely infectious diseases transmissible by respiratory aerosolization, direct contact with primary body fluids, or dried infectious particles from body fluids. The SCSU has three patient rooms plus an occupational-exposure isolation room (OETR). It is equipped to meet all clinical-care needs, including basic medical observation, minor surgical procedures, and intensive care. Thus, planning had to address housekeeping, security, emergency evacuation, and the use of experimental therapeutics.

This article will discuss design crite-

ria used for the SCSU, wruch can be applied for any other type of airborne-infection-isolation suite (AilS).

General Design Features

The SCSO receives air via the main clinical supply-air system, wruch is served by air-handling units located in another part of the building. Trus is a manifolded system providing supply air to all patient rooms and support spaces (Figure 1).

All clinical spaces are served by a non-recirculating (100-percent exhaust) HVAC system. All general clinical spaces, as weiJ as the adjacent Vaccine Evaluation Clinic (VEC), are served by general exhaust systems consisting of multiple exhaust fans connected to a common exhaust-air manifold. Dedicated isolation-room exhaust systems consist of multiple exhaust fans connected to a common exhaust-air manifold. If one fan fails, the remaining fans



FIGURE 1 General arrangement of mechanical systems

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filtration unit

compensate to provide the required design exhaust-air quantity.

The VEC is exhausted through the general clinical exhaust system, while the SCSU is exhausted through a dedicated system. Exhaust air from all SCSU spaces, including the OEIR is manifolded in the dedicated exhaust system.

The dedicated system is equipped with high-efficiency-particulate-air (HEPA) filters prior to exhaust fans and discharges outside (Figure 2).

At both entrances to the SCSU are vestibules maintained at negative pressure relative to the adjoining corridors. The OEIR has an anteroom maintained at negative pressure relative to the rest of the SCSU to mitigate the risk of release of an airborne agent. The negative pressure of the patient rooms, the OEIR, and the SCSU is monitored and alarmed at the nurses' station.



FIGURE 2. Dual-use-isolation-room anteroom mechanical systems.

SCSU Design Features

Design features specific to the SCSU include:

• Walls (outside of the OEIR) sealed to minimize uncontrolled

air movement to and from adjacent spaces, sealed piping penetrations, and openings in electrical boxes and other component device boxes sealed with caulking.



DESIGN OF A SPECIALIZED AIRBORNE-INFECTION-ISOLATION SUITE



FIGURE 3. Dual-use Isolation room.

- Self-closing exit doors.
- Balanced ventilation air to main- tain negative pressure (inflow of air) relative to the corridor (between public corridor and the SCSU).
- Exhaust-air grilles located low on patient-room walls.
- Pressure-monitoring system continuously indicating airflow from adjacent spaces into the SCSU.
- Displays of relative space pressurization at the corridor and vestibuJes.
- · Remote alarms at the nurses'
- station.
- Twelve-air-changes-per-hour (ACH) minimum ventilation rate of non-recirculating air (100- percent outdoor air) for the SCSU and OEIR.
- Dedicated toilet, bath, and handwashing facilities in each patient room.
- Dedicated exhaust-air system serving the SCSU and OEIR.

The dedicated constant-volume exhaust system consists of three variable-speed exhaust fans. Each fan is designed at 50 percent of capacity to provide the required level of redundancy and reliability and 20-percent reserve-airflow capacity to account for final air-balance adjustments and future needs. The fans operate continuously, their speed controlled with variable-frequency drives based on a static-pressure sensor in the exhaust ductwork. Discharge from the fans is manifolded to ensure a high stackdischarge velocity.

One of the exhaust fans receives life-safety branch power; t.he other two receive general standby-equipment branch power. In the event of normal power loss, life-safety power is restored within 10 sec, and the first fan is re-engaged and ramps up to satisfy required minimum exhaust and directional airflows. Shortly thereafter, power to the other fans is restored, and the fans are re-engaged. Air distribution to individual spaces/ rooms is by constant- and/or vari- ablevolume, pressure-independent supply and exhaust air terminals to each temperature- and/or pressure- control zone. Pairs of supply and ex- haust venturi-style air-terminal units track airflow to ensure the specified pressurization (directional airflow) is maintained. The onJy prerequisite for the air terminals to operate properly is sufficient duct static pressure. This is monitored via differential-pressure switches across each air valve. Additionally. automatic bubble-tight isolation dampers are provided in the supply duct to each zone.

All exhaust air-terminal units are low-leakage (less than 3 cfm at 1 in. water), with high-speed actuators WHAT MAKES THE CBEX DIFFERENT FROM ANY OTHER BOILER IN THE WORLD?

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(less-than-1-sec fuiJ response time) to stay within 5 percent of airflow set point. The air-terminal units' controllers are linked between supply and exhaust. If duct static pressure falls below levels required for accurate tracking, the supply-air bubble-tight dampers close, and the exhaust-air valves are commanded to a minimum position, maintaining directional airflow.

The control devices operate autonomously, even in the event communications with the overall building control system are disrupted. All of the controllers and bubble-tight dampers are provided uninterruptible power to ensure the air-terminal units track each other to maintain the specified airflow differential. AJI serviceable components, such as air terminals, exhaust valves, and reheat coils, are located in the interstitial space outside of the patient-care/containment area.

All exhaust ductwork for the SCSU and negative-pressure examination room on the room side of the HEPA filter is made of welded stainless steel (Figure 3). The supply ductwork is

made of galvanized steel augmented with welded stainJess steel. Bubbletight dampers are provided on all supply ducts branching from the

interstitial level to SCSU patient rooms and the OEIR. These are fast-acting pneumatic actuators that close upon a loss of exhaust airflow. Ductwork from the bubbletight dampers to air devices is made

Backtlow preventers are installed in domestic-water piping running to the SCSU. When the SCSU is in isolation mode, oxygen, medical vacuum, and medical air are provided via local dedicated units. HEPA filtration is provided on the main plumbing vents leaving the SCSU.

Emergency power is provided for critical branch, life-safety, and standby equipment in the SCSU and for exhaust fans on the interstitial level as appropriate.

Electrical services for the OEIR

are sealed airtight to mm1mize outside penetrations. A third inner layer of sheetrock prevents air migration that can occur as the result of required utility boxes and associated penetrations.

Room Control Sequences

Under normal conditions, supply- and ex haust-air valves oper-



FIGURE 4. Manifolded supply-air systems and dedicated exhaust-air system for the SCSU



FIGURE 5. ManHolded supply alr system and dedicated exhaust-air system for the SCSU.

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ate at "constant" airflow set points. The fixed differential between supply- and exhaust-airflow set points provides directional airflow and room pressurization. For example, an exhaust-airflow set point of 700 cfm and supply-airflow set point of 600 cfm provides 100-cfm inward airflow, maintaining a room at negative pressure. As long as the "measured" differential pressure across the supply- and exhaust-air valves is within an acceptable range (0.6 in. wg to 3.0 in. wg), air valves maintain their respective constantairflow set points, maintaining directional airflow and room pressurization. Differential-pressure switches across each supply- and exhaust-air valve continuously monitor airflow.

If the differential pressure across either the supply- or exhaust-air valve in any room drops below 0.3 in. wg, the bubble-tight damper on



F1GURE 6. Patient-room airflow.

the supply duct to that room closes immediately. The supply-airflow and exhaust-airflow set points are reset to emergency-mode minimum values. When the main supply-duct static pressure and main exhaust-duct static pressure J;se above 0.85 in. wg, the bubble-tight damper opens, sup-

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ply- and exhaust-airflow set points are reset to their normal values, and the room reverts to normal operation. The room control sequences are illustrated in figures 4-6.

Particle Dynamics

Clinically applicable distinctions are made between short-range airborne-infection routes (between individuals less than 3.28 ft apart) and long-range routes (within a room, between rooms, or between individuals greater than 3.28 ft apart). Small droplets may participate in short-range transmission, but are more Jjkely than large droplets to evaporate to become droplet nuclei. True long-range aerosol transmission becomes possible when droplets of infectious material are small enough to remam airborne almost indefinitely and be transmitted over long distances. There is



FIGURE 7 SCSU ventilation system.

essential agreement that particles with an aerodynamic diameter of 5 J.Iffi or less are aerosols, while particles with an aerodynantic diameter of 20 pm are large droplets.

Methodology

For the SCSU, computational flilld dynamics and particle tracking were

used to study the influence of ventilation flow rate (12 ACH and 16 ACH), exhaust location (high exhaust, low exhaust, and the combination ofhlgh and low exhausts), and patient position (sitting and lying) on the removal of aerosol generated by a cough.

The model sillte consisted of three rooms-main isolation room with



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Case	Patient	ACH	Exhaust	Results: 3 min				Results: 5 min			
				Vented	Bathroom	Room	Breathing	Vented	Bathroom	Room	Breathing
1	Sitting	12	Low	22%	24%	54%	9%	35%	23%	42%	6%
2	Lying	12	Low	21%	2%	77%	7%	29%	4%	67%	7%
3	Sitting	16	Low	27%	7%	66%	6%	40%	7%	53%	5%
4	Lying	16	Low	26%	4%	71%	8%	35%	6%	60%	7%
5	Sitting	12	L/H	15%	19%	66%	12%	26%	20%	55%	10%
6	Lying	12	L/H	13%	1%	85%	11%	24%	3%	73%	11%
7	Sitting	16	L/H	16%	18%	66%	13%	30%	20%	50%	8%
8	Lying	16	L/H	15%	1%	84%	8%	25%	4%	71%	14%
9	Sitting	12	High	13%	19%	68%	14%	24%	21%	55%	10%
10	Lying	12	High	45%	3%	52%	6%	52%	5%	43%	7%
11	Sitting	16	High	24%	14%	62%	13%	37%	17%	46%	8%
12	Lying	16	High	21%	1%	78%	9%	34%	2%	64%	10%

patient bed, bathroom, and anteroom-connected via door gaps.

A patient's cough was assumed to release 3,000 droplet nuclei with diameters of 3 IJm.^{2,3} Further, the particles were assumed to be 93°F when released from the mouth.⁴

Results

Three thousand particles were tracked individually for 5 min, with the number of particles removed from the room recorded every minute. The location of each particle remaining in the room was recorded. Particular attention was paid to the number of particles remaining in the breathing zone of the main room and the breathing zone around the bed where health-care workers were most likely to be. The breathing zone of the main room was defined as the space 3.6 ft to 5.75 ft from the floor; the breathing zone around the bed was defined as the space 9 in. from the edges of, and 3.6 ft to 5.75 ft above, the bed (Figure 7).

Table 1 shows the percentage of particles removed from the suite 3 min and 5 rrtin after a cough and the percentages of particles remaining in the bathroom, the main room, and the breathing zone of the main room. During the first 3 min, the system removed 13 percent to 45 percent of the 3,000 particles. Within 5 min, the percentage of particles removed increased to 25 to 55 percent, which is consistent with estimates⁵ that, assuming a well-mixing condition,

99 percent of particles will be removed after 9 min at 16 ACH. Note the percentage of particles in the bathroom is much higher when a sitting patient coughs than when a lyjng patient coughs. The reason is the distribution of particles in the air current and the location of the bathroom.

Figure 8 shows the number of particles removed 3 min and 5 min after the cough of a sitting patient and a lying patient. The combination of low and high exhausts was least effective in removing particles. Generally, with the exception of Case 10, the most particles were removed with low exhaust and high flow. With a lying patient, 12 ACH (Case 10) removed more particles than 16 ACH



FIGURE 8.Number of particles removed 3 min and 5 min after a cough.

(Case 12) with high exhaust. This can be explained by the flow pattern above the patient, determined by the downward forced convection from the ceiling diffusers above the bed and the upward flow of the cough from the mouth of the lying patient.

With greater downward flow (16 ACH, rather than 12 ACH) from the ceiling diffuser, the upward movement of droplets carried by the cough of a lying patient could be suppressed and, thus, have difficulty reaching high exhausts.

For a sitting patient, particles in the breathing zone generally were reduced from minutes 3 to 5 (Figure 9). For a lying patient, they sometimes increased, depending on the ventilation system and flow rate. Figure 10 shows the combination of low and high exhausts resulted in a higher number of particles in the breathing zone around the bed, especially in the cases (6 and 8) of a lying patient. High exhausts with a high flow rate (Case 12) did not remove particles around the bed effectively. Figure 10 indicates fewer particles around the bed with low exhausts.

The number of particles below the breathing zone was higher with low exhausts than with high exhausts (Figure 11). High exhausts with high flow rates (Case 12) did not seem to remove particles around the bed effectively. The number of particles above the breathing zone was higher with a lying patient regardless of flow rate and exhaust location (Figure 12). This was attributed primarily to the initial upward momentum of the cough jet.

Conclusions

Conclusions that can be drawn from the study include:

• Low exhaust outperforms other exhaust locations in terms of particle removal and number of particles remaining around a bed.

• Increasing ventilation flow from 12 to 16 ACH generally helps to remove particles from an isolation





FIGURE 9. Number of particles remaining In breathing zone.

FIGURE 10. Number of particles remaining In breathing zone around bed.







FIGURE 12. Number of particles above breathing zone.

room (except in cases of high exhaust and a lying patient coughing), but not necessarily a breathing zone.

• The number of particles in a bathroom resulting from a cough in a main room is dependent on air-current particle distribution and bathroom location.

Standard operating procedures are extremely important and should be developed as part of the planning process, with consideration given to facility purpose and features.

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